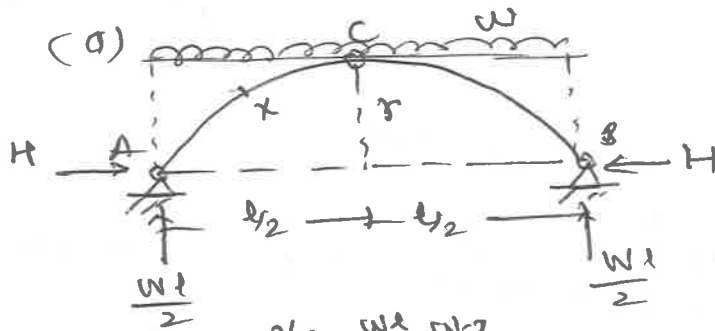


Answer Key (Model Answer)
By Prof. D.M. Joshi SOE,
(9324866015)

MAX MARKS-80
(3 hours)

Q1 Compulsory (Any four 5x4=20)

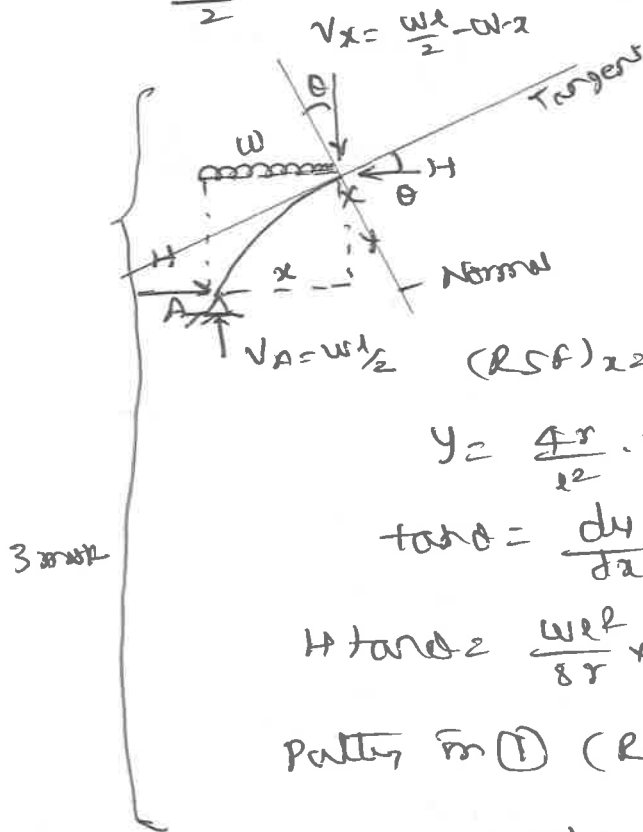


$\sum M = 0$ about C

$$H \times r + W \frac{l}{2} \cdot \frac{l}{4} = \frac{Wl}{2} \times \frac{l}{2}$$

$$H = \frac{Wl^2}{8r}$$

2 marks



$$H = \frac{Wl^2}{8r} \leftarrow$$

$$V_x = \frac{W}{2} (1-2x) \downarrow$$

$$(RSF)_x = H \sin \theta - V_x \cos \theta$$

$$(RSF)_x = \cos \theta \left[H \tan \theta - V_x \right] \quad \text{--- (1)}$$

$$y = \frac{4r}{l^2} \cdot x(1-x)$$

$$\tan \theta = \frac{dy}{dx} = \frac{4r}{l^2} (1-2x)$$

$$H \tan \theta = \frac{Wl^2}{8r} \times \frac{4r}{l^2} (1-2x) = \frac{W}{2} (1-2x)$$

Putting in (1) $(RSF)_x = \cos \theta \left[\frac{W}{2} (1-2x) - \frac{W}{2} (1-2x) \right]$

$$(RSF)_x = 0$$

Here proved

Total 2+3=5 marks

Q1(b)

Prof Perry's formula

$$\left(\frac{\sigma_{max}}{\sigma_a} - 1 \right) \left(1 - \frac{\sigma_a}{\sigma_p} \right) = \frac{1.2 \cdot e \cdot y_c}{k^2} \quad \text{--- 100 marks}$$

Term Explanation --- 2 marks

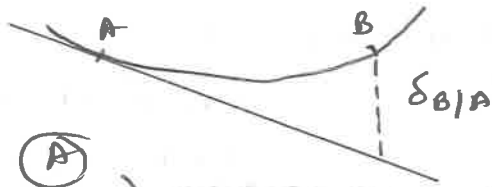
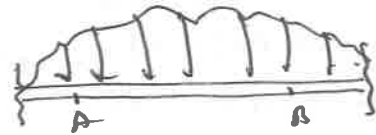
Important

P can be determined if σ_{max} & σ_a are given
Not possible with Secant formula

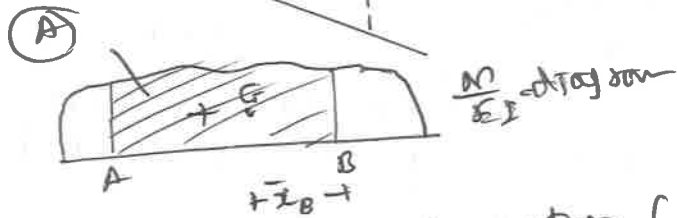
--- 2m

--- 100 marks

Moment Area II theorem



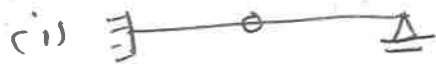
$$\delta_{B/A} = \int_A^B (A \bar{x}_B) \frac{M}{EI} dx$$



Statement + Explanation (1+2) marks

Real Beam

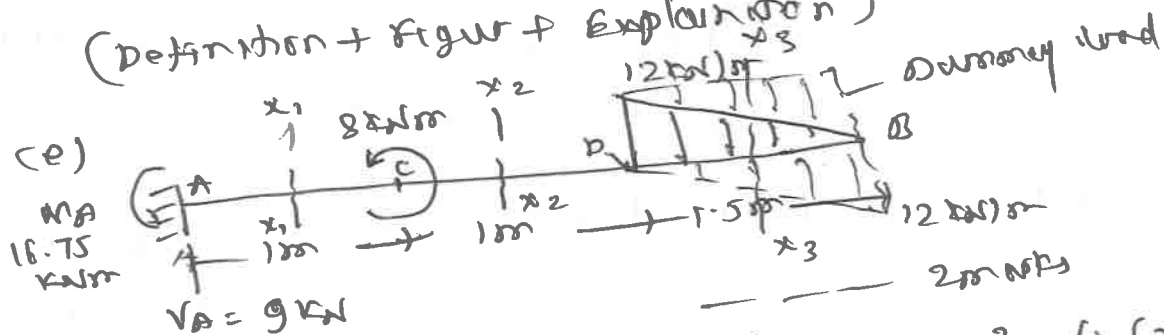
conjugate beam



0.5 x 4 = 2 marks

(d) Unsymmetrical Bending --- 3 marks
shear centre --- 2 marks

(Definition + figure + Explanation)



$$M_x = 9x - 16.75 \left\{ \begin{array}{l} - 8(x-1)^0 \\ - 6(x-2)^2 + \frac{4}{3}(x-2)^3 \end{array} \right\}$$

0.5 marks

(f) (i) Principle of superposition — 2 m

(ii) Castigliano's theorem — 3 m

3/8

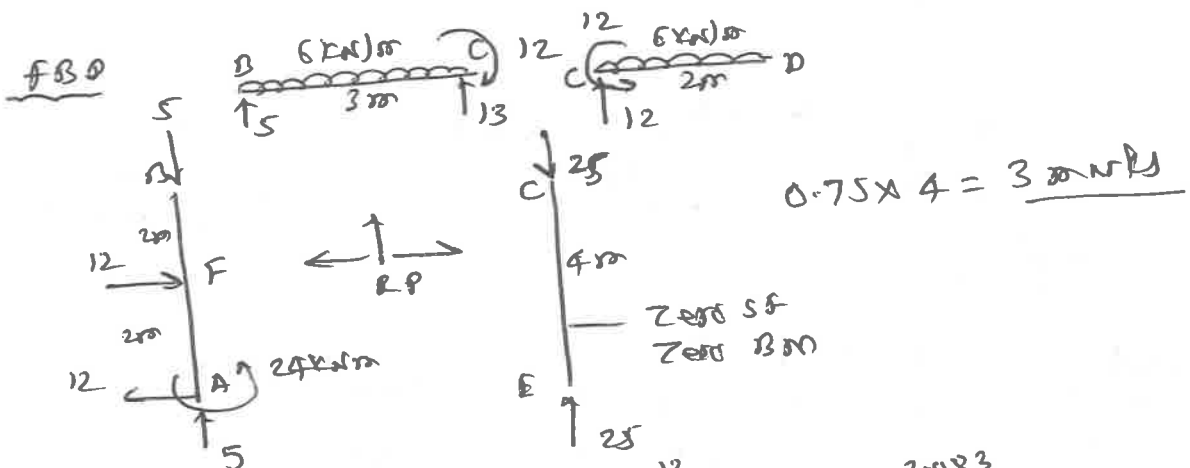
(~~statement~~ statement + figure + explanation)

Q 20a)
10 marks

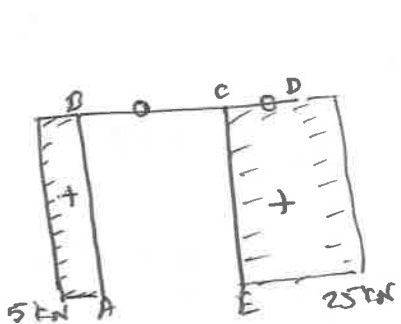
$V_E = 25 \text{ kN} \uparrow$ $V_A = 5 \text{ kN} \uparrow$

$H_A = 12 \text{ kN} \leftarrow$ $M_A = 24 \text{ kNm}$

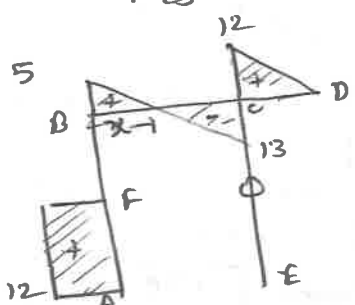
2 marks



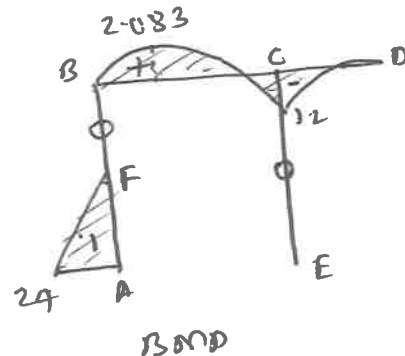
$0.75 \times 4 = 3 \text{ marks}$



AFD Composite



SFD $x = \frac{5}{6} \text{ m}$



BMD

1 mark

+

2 marks

+

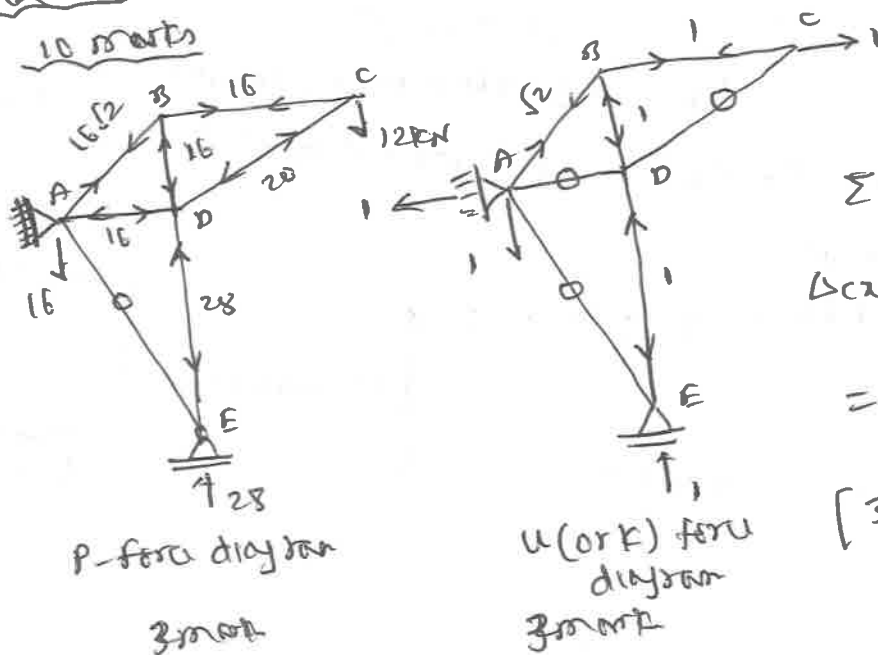
$2 \text{ marks} = 5 \text{ marks}$
 $2 + 3 + 5 = 10 \text{ marks}$

Q 20b)

10 marks

$\Delta_{Cx} = ? \rightarrow$

$AE = 30000 \text{ kN}$



P-force diagram

3 marks

u (or k) force diagram
3 marks

Table - 3 marks

$\sum P u_i = 359.765$

$\Delta_{Cx} = \frac{\sum P u_i}{AE}$
 $= \frac{11.992 \text{ mm}}{1000} \rightarrow$

$[3 + 3 + 3 + 1 = 10 \text{ marks}]$

Q 3(a)
10 marks

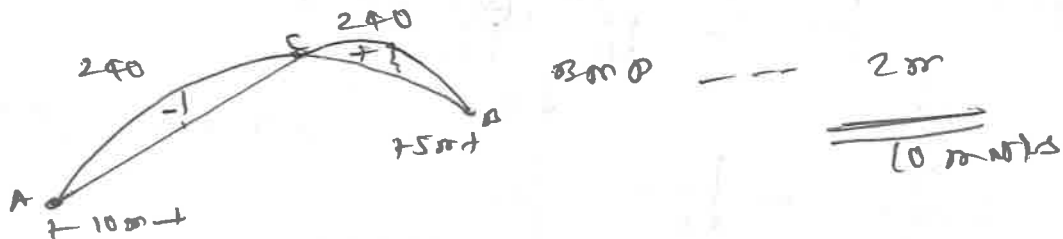
(i) $y = kx^2$ (C-origins)

4/8

$\frac{h_1}{h_2} = \frac{x_1^2}{x_2^2} \Rightarrow h = 1.5m \quad \text{---} \quad 1m$

(ii) $V_A = 48kN \quad V_B = 84kN \quad (\uparrow) \quad \left. \vphantom{V_A} \right\} \text{---} 4m$
 $H = 160kN \quad (\rightarrow \leftarrow)$

(iii) for AC position $x = 10m \quad M_{max} = -240kNm$
 for BC position $x = 5m \quad M_{max} = 240kNm \quad \left. \vphantom{M_{max}} \right\} \text{---} 3m$



Q 3(b)
10 marks

$l_c = 0.5L = 3m$

$A = \frac{\pi}{4} (200^2 - 150^2) = 13.744 \times 10^3 \text{ mm}^2$

$I = \frac{\pi}{64} (200^4 - 150^4) = 53.689 \times 10^6 \text{ mm}^4$

$Z = \frac{I}{y_c} = 536.89 \times 10^3 \text{ mm}^3$

$\sigma_a = \frac{180 \times 10^3}{13.744 \times 10^3} = 13.097 \text{ N/mm}^2 \text{ (Comp)}$

$\theta = \left[\frac{3m}{2} \sqrt{\frac{180 \times 10^3}{96 \times 10^3 \times 53.689 \times 10^6}} \right] \times \frac{180}{\pi}$

$\theta = 16.06^\circ$

$m = P \cdot e \cdot \sec \theta = 180 \times 10^3 \times 45 \times \sec \theta$

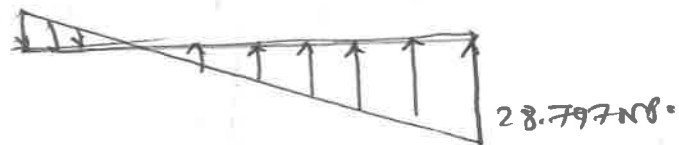
$m = 8.429 \times 10^6 \text{ Nm}$

$\sigma_b = \pm \frac{8.429 \times 10^6}{536.89 \times 10^3} = \pm 15.700 \text{ N/mm}^2$

$\sigma_{max} = \sigma_a + \sigma_b = 28.797 \text{ MPa (Comp)}$

$\sigma_{min} = \sigma_a - \sigma_b = 2.603 \text{ MPa (Tensile)}$

2.603 MPa

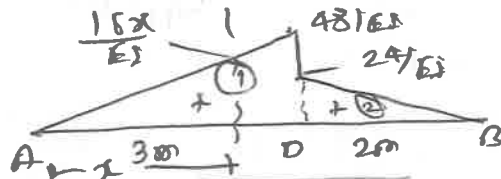
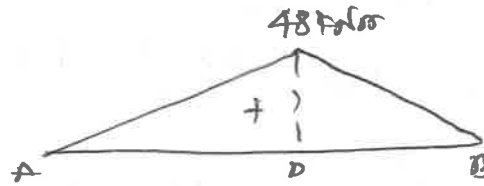
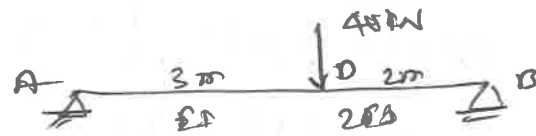


200mm

10 marks

Q 400) Moment Area method

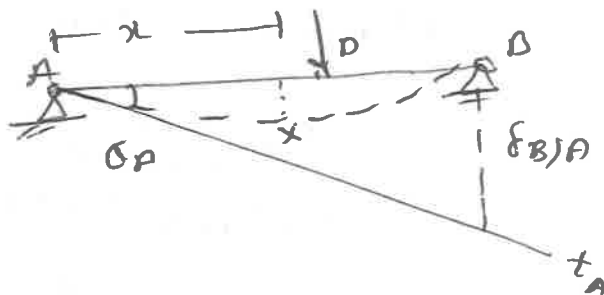
5/8



Area	\bar{x}_A	\bar{x}_B
$A_1 = \frac{72}{EI}$	2	3
$A_2 = \frac{24}{EI}$	$\frac{11}{3}$	$\frac{4}{3}$

BMD

$\frac{M}{EI}$ diagram



Theorem I $\delta_{B/A} = \frac{72}{EI} \times 3$

$$\delta_{B/A} = \sum_A [A \bar{x}_B] \frac{m}{EI} = \frac{72}{EI} \times 3 + \frac{24}{EI} \times \frac{4}{3} = \frac{248}{EI}$$

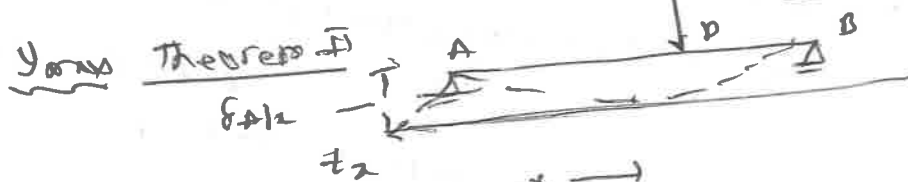
$\theta_{B/A} = \theta_A = \frac{\delta_{B/A}}{BA} = \frac{49.6}{EI}$ (2) --- 5 marks

Slope at x in zero Theorem I $\delta_{A/x} = \sum_A [A] \frac{m}{EI}$

$\theta_1 - \theta_2 = \frac{1}{2} \cdot x \cdot \frac{16x}{EI}$

$8x^2 = 49.6$

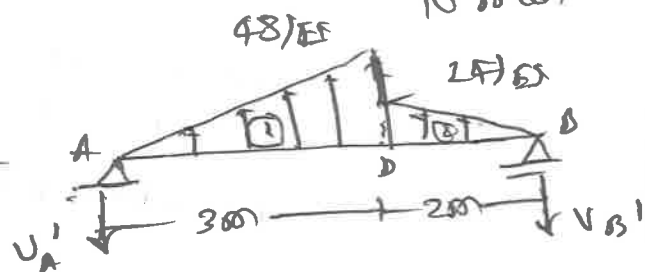
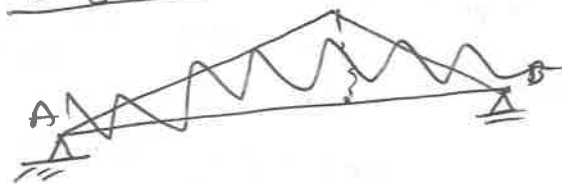
$x = \frac{2.49 \text{ m} - \text{Ans}}{2 \text{ m mark}}$



$y_{\text{max}} = \delta_{A/x} = \sum_A (A \bar{x}_A) \frac{m}{EI} = \frac{82.333}{EI} \text{ m} - \text{Ans}$

--- 2 marks

Conjugate beam method



loads	\bar{x}_A	\bar{x}_B
$P_1 = \frac{72}{EI} \uparrow$	2m	3m
$P_2 = \frac{24}{EI} \uparrow$	$\frac{11}{3}$	$\frac{4}{3}$

$$\Sigma P = \frac{96}{EI} \uparrow$$

$$\Sigma V = 0 \quad V_B' = \frac{46.4}{EI} \uparrow$$

$$\Sigma M = 0 \text{ about } B$$

$$V_A' \times 5 = \frac{72}{EI} \times 3 + \frac{24}{EI} \times \frac{4}{3}$$

$$V_A' = \frac{49.6}{EI} \downarrow$$

$$(O_A)_{\text{real}} = (SF_A)_{\text{orig}} = -\frac{49.6}{EI}$$

$$\theta_A = \frac{49.6}{EI} \downarrow$$

$$(O_D)_{\text{real}} = (SF_D)_L = -\frac{49.6}{EI} + \frac{72}{EI} = \frac{22.4}{EI} \uparrow$$

$$\therefore \theta_{\text{real}} = 0 \text{ betw } A \text{ \& } D$$

$$(O_x)_{\text{real}} = (SF_x)_{\text{orig}} = -\frac{49.6}{EI} \cdot x + \frac{8x^2}{EI} = 0$$

$$\therefore x = 2.49 \text{ m}$$

$$\therefore (Y_{\text{max}})_{\text{real}} = (BM_x)_{\text{orig}} = -\frac{49.6}{EI} \times 2.49 + \frac{8}{EI} \times \frac{(2.49)^2}{3}$$

$$Y_{\text{max}} = -\frac{82.335}{EI}$$

$$Y_{\text{max}} = \frac{82.335}{EI} \text{ m } \downarrow$$

Q 4(b) $R = \Sigma W = 90 \text{ kN}$ $l_w = 10 \text{ m}$

$$\bar{x}_L = \frac{(20 \times 3 + 20 \times 5 + 25 \times 7 + 15 \times 10)}{90} = 5.389 \text{ m}$$

middle 20 kN load is critical load 'P'

$$d = x_w - x_p = 5.389 - 5 = 0.389 \text{ m}$$

$$\therefore x = \frac{d}{2} = 0.195 \text{ m}$$

critical section $x_c = 12 - 0.195 = 11.805$

from left $x_w = 12 + 0.195 = 12.195$

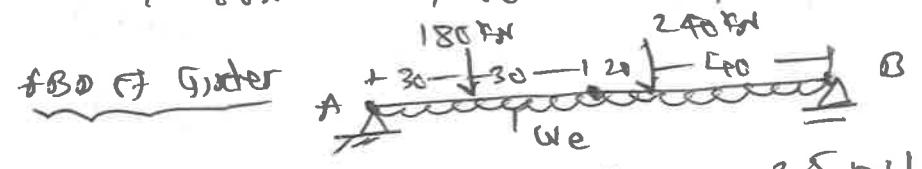
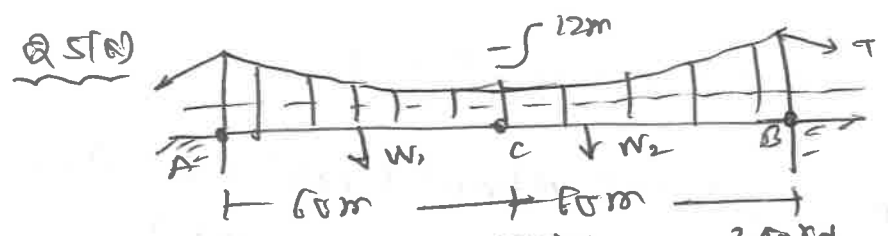
for above placement $V_A = 44.209 \text{ kN}$
 $V_B = 45.791 \text{ kN}$

$(BM)_{\text{critical}} = \frac{432.596 \text{ kNm}}{\text{critical load}}$

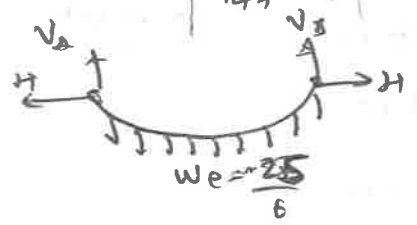
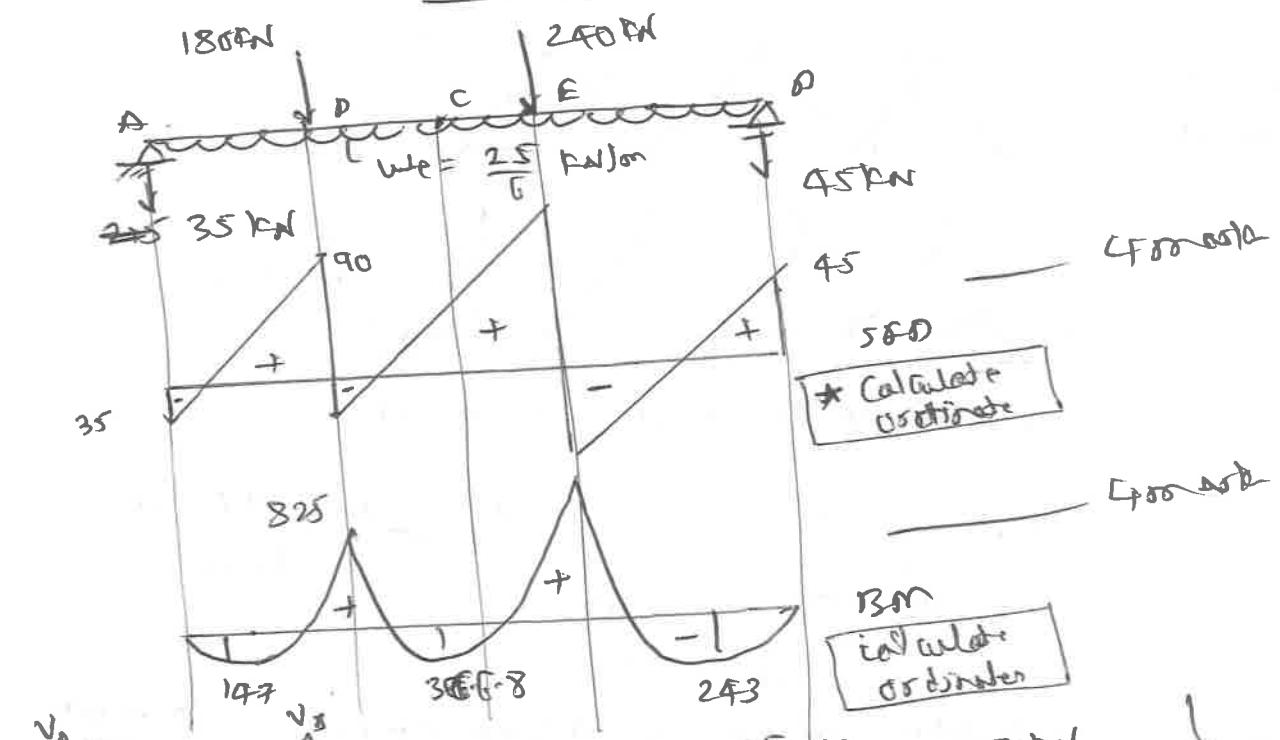
under P_L

$P (= 20 \text{ kN})$

2m
10 marks

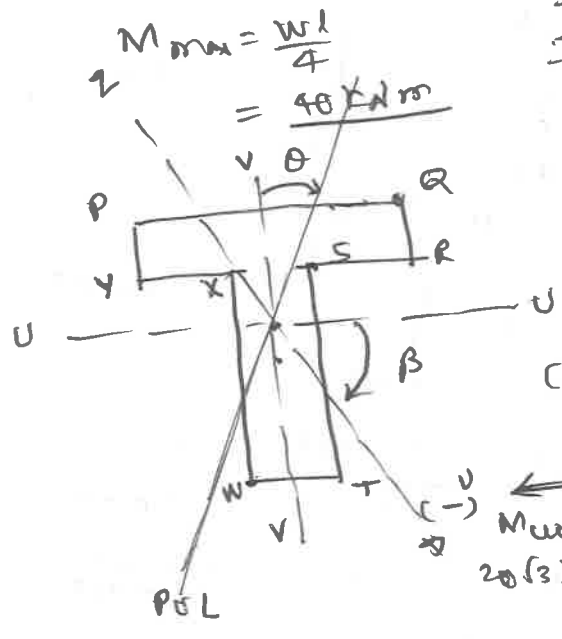


$w_e = 4.167 \text{ kN/m} = \frac{25}{6} \text{ kN/m} \quad \text{--- 2 marks}$



$V_A = V_B = V = \frac{25 \times 12}{2} = 250 \text{ kN}$
 $H = \frac{w_e l^2}{8d} = 750 \text{ kN} = T_{min} = T_c$
 $T_{max} = \sqrt{V^2 + H^2} = 790.569 \text{ kN}$ } --- 2 marks

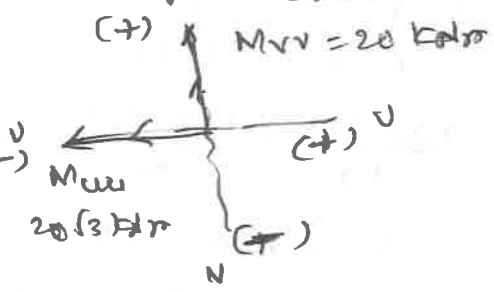
Q5(d) T-section 8 marks



$\bar{y} = y_{top} = 70 \text{ mm}$ $y_{bottom} = 130 \text{ mm}$
 $I_{uu} = 24.1 \times 10^6 \text{ mm}^4 (= I_{xx})$
 $I_{vv} = 3 \times 10^6 \text{ mm}^4 (= I_{yy})$

$\tan \beta = \frac{I_{vv}}{I_{uu}} \tan \theta$
 $\beta = 77.882^\circ (\downarrow)$

$\sigma_{max} = \sigma_Q$ (comp)
 $\sigma_{min} = \sigma_W$ (tensile)



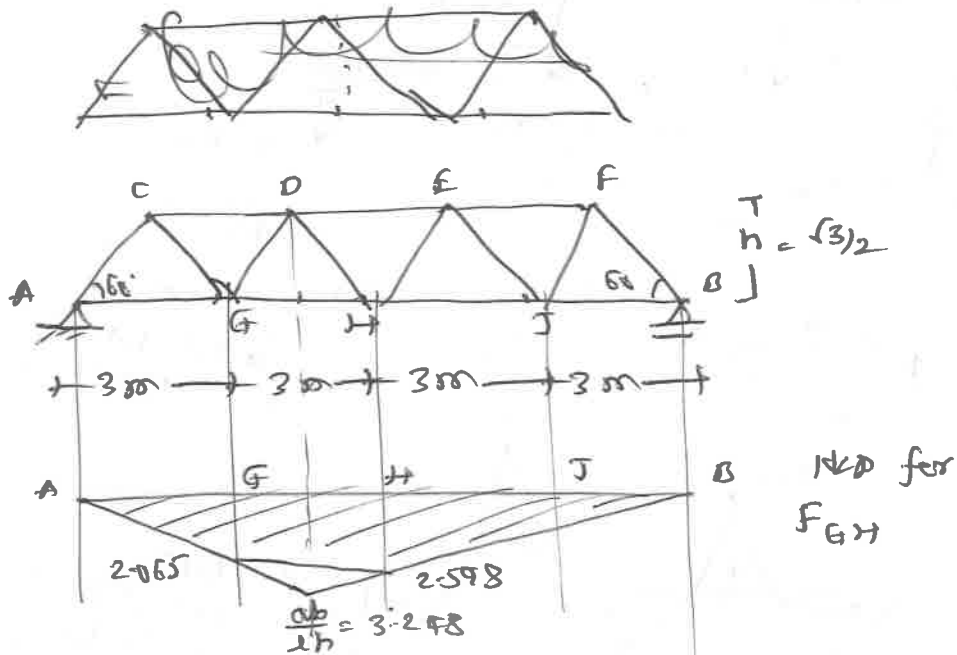
$\sigma_Q = 500.28 \text{ (N/mm}^2 \text{)} \text{ (comp)}$
 --- 2 marks

8/8

$\sigma_{W2} = 252.755 \text{ N/mm}^2$ (Correct)

OR $\sigma_{W2} = 252.755 \text{ N/mm}^2$ (Tensile) — 4m
 2 marks
 8 marks

Q 6(a)
4 marks

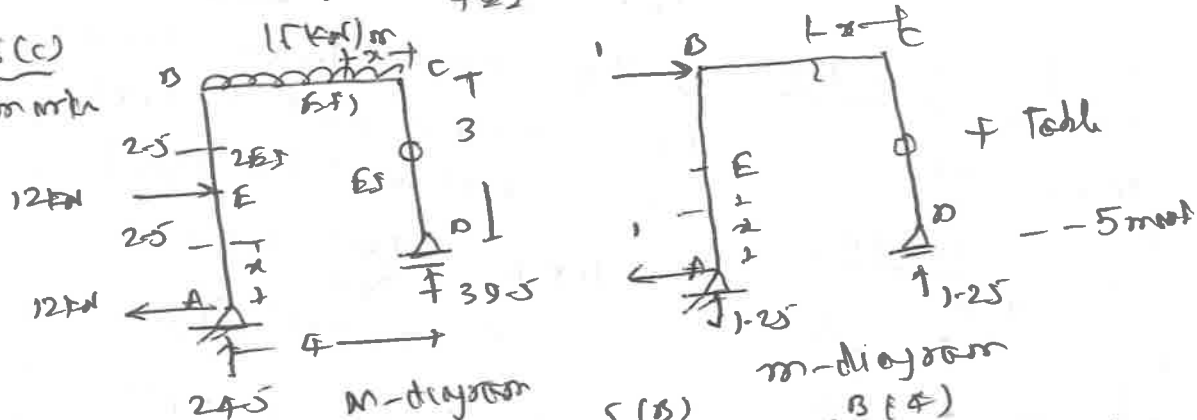


Calculation 2 marks + diagram 2 marks = 4 marks

Q 6(b)
6 marks

$U = U_{bending} = \frac{\pi W R^3}{8 E}$ — 4m
 $\Delta_{By} = \frac{\pi W R^3}{4 E}$ — 2m

Q 6(c)
10 marks



$\Delta_{Bx} = \int \frac{M_x dx}{E I} = \int_{A(0)}^{E(2.5)} + \int_{2.5(E)}^{5(B)} + \int_{C(0)}^{D(4)}$ — 4 marks
 $= \frac{3 \cdot 2.5}{E I} + \frac{140.625}{E I} + \frac{4 \cdot 3 \cdot 333}{E I} = \frac{585.208}{E I}$ — 1 mark

$\Delta_{Bx} = 14.63 \text{ mm} (\rightarrow)$ — 4m
 10 marks